

#### **Final Project**

# Damage Identification of Cylindrical Shell Structures

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# Layout

#### **1. Introduction**

2. Theoretical Analysis

3. Data Analysis and Results





#### Introduction

The advent of smart materials

**Electromechanical (EM) method** 

- Interaction relation between smart material transducer and host structure
- Impedance analyzer



### Introduction





## Introduction

## Piezoelectricity

## **Direct Piezoelectric Effect**

#### Stress field >> Electric charges >> Sensors

#### **Converse Piezoelectric Effect**

Electric Field > Strain field > Actuators



#### Introduction

Piezoelectric constitutive relations (IEEE standard, 1987)







#### Theoretical Analysis

Modeling of PZT Transducers

≻2-D generic model

Modeling Of Host Structure

≻Cylindrical shell

➤Classical theory of thin shell

≻P-Ritz Method



### Modeling of PZT Transducer





#### Theoretical Analysis

#### Motion equation

$$\frac{\widetilde{Y}_{p}^{E}}{1-\mu_{p}^{2}}\frac{\partial^{2}u_{p}}{\partial x_{p}^{2}} = \rho_{p}\vec{u}_{p}$$
$$\frac{\widetilde{Y}_{p}^{E}}{1-\mu_{p}^{2}}\frac{\partial^{2}v_{p}}{\partial y_{p}^{2}} = \rho_{p}\vec{v}_{p}$$

$$u_{p} = (A \sin K_{p} x_{p} + B \cos K_{p} x_{p})e^{j\omega t}$$

$$v_{p} = (C \sin K_{p} y_{p} + D \cos K_{p} y_{p})e^{j\omega t}$$

#### force applied on PZT patch

stresses



constitutive equation

$$D_{3} = \{d_{31} \quad d_{32}\} \begin{bmatrix} T_{1} \\ T_{2} \end{bmatrix} + \widetilde{\varepsilon}_{33}^{T} E_{33}$$





#### Theoretical Analysis

#### EM admittance of PZT transducers





#### **Theoretical Analysis**

Cylindrical shell with a pair of PZT patches





#### Theoretical Analysis

#### **Classical Theory of Thin Shell**

$$\varepsilon_{11} = \frac{1}{A_1} \frac{\partial}{\partial \alpha_1} (\overline{u}_1 + \alpha_3 \beta_1) + \frac{1}{A_1 A_2} \frac{\partial A_1}{\partial \alpha_2} (\overline{u}_2 + \alpha_3 \beta_2) + \frac{\overline{u}_3}{R_1}$$
$$\varepsilon_{22} = \frac{1}{A_2} \frac{\partial}{\partial \alpha_2} (\overline{u}_2 + \alpha_3 \beta_2) + \frac{1}{A_1 A_2} \frac{\partial A_2}{\partial \alpha_1} (\overline{u}_1 + \alpha_3 \beta_1) + \frac{\overline{u}_3}{R_2}$$

$$\varepsilon_{12} = \frac{A_2}{A_1} \frac{\partial}{\partial \alpha_1} \left( \frac{\overline{u}_2 + \alpha_3 \beta_2}{A_2} \right) + \frac{A_1}{A_2} \frac{\partial}{\partial \alpha_2} \left( \frac{\overline{u}_1 + \alpha_3 \beta_1}{A_1} \right)$$
$$\overline{u}_1 = \frac{1}{2} \frac{\partial \overline{u}_2}{\partial u}$$

$$\beta_{1} = \frac{\alpha_{1}}{R_{1}} - \frac{1}{A_{1}} \frac{\partial \alpha_{3}}{\partial \alpha_{1}}$$

$$\beta_2 = \frac{\overline{u}_2}{R_2} - \frac{1}{A_2} \frac{\partial \overline{u}_3}{\partial \alpha_2}$$





#### Theoretical Analysis

# Displacement relations for cylindrical shell under pure bending

$$u_{x} = -(r - a) \frac{\partial w}{\partial x}$$
$$v_{\theta} = -\frac{(r - a)}{a} \frac{\partial w}{\partial \theta}$$
$$w = \overline{w}(x, \theta)$$

#### **Strain-displacement relations**

$$\varepsilon_{xx} = -(r-a)\frac{\partial^2 \overline{w}}{\partial x^2}$$
$$\varepsilon_{\theta\theta} = \frac{1}{a}\overline{w} - \frac{1}{a^2}(r-a)\frac{\partial^2 \overline{w}}{\partial \theta^2}$$
$$\varepsilon_{x\theta} = -\frac{2}{a}(r-a)\frac{\partial^2 \overline{w}}{\partial x \partial \theta}$$

#### **Stress-strain relations**

$$\sigma_{xx} = \frac{E}{1 - \mu^2} (\varepsilon_{xx} + \mu \varepsilon_{\theta\theta})$$
$$\sigma_{\theta\theta} = \frac{E}{1 - \mu^2} (\varepsilon_{\theta\theta} + \mu \varepsilon_{xx})$$
$$\sigma_{x\theta} = G\varepsilon_{x\theta}$$



Hamilton's principle

$$\delta \int_{t_0}^{t_1} (T - U + W_{nc}) dt = 0$$

#### Kinetic energy of cylindrical shell

$$T = \frac{1}{2} \rho \iiint (\dot{u}_x^2 + \dot{v}_\theta^2 + \dot{w}^2) ad \,\theta dr dx$$

The strain energy of cylindrical shell

$$U = \iiint \frac{1}{2} (\sigma_{xx} \varepsilon_{xx} + \sigma_{\theta\theta} \varepsilon_{\theta\theta} + \sigma_{x\theta} \varepsilon_{x\theta}) a d\theta dx dr$$



> non-dimensionalization 
$$\xi = \frac{2x}{L} - 1$$
  $\eta = \frac{2\theta}{\pi} - 1$   
> displacement  $W = \sum_{q=0}^{N} \sum_{i=0}^{q} c_m \phi_m$ 

 $\phi_m$  satisfies the following conditions

- linearly independent
- Form a complete system of functions
- Satisfy the geometrical boundary conditions

#### > The polynomial shape functions

$$\phi_{m} = \xi^{i} \eta^{q-i} \phi_{0} \quad \text{m=1,2,...,} \quad \frac{(N+1)(N+2)}{2}$$
$$\phi_{0} = \prod_{p} \left[ \Gamma_{p} \left(\xi, \eta\right) \right]^{\theta_{p}}$$



## The Hamilton's principle





# Results

## **Comparison of Theoretical Results for Damaged and Undamaged Shell Specimen**



Undamaged	Damaged	Shift
30. 891	30. 889	0.002
31.36	31. 313	0.047
32.009	32.002	0.007
32. 649	32. 627	0. 022
33. 562	33. 555	0.007
34. 168	34. 124	0.044
35.95	35. 946	0.004
36. 137	36. 137	0
36. 807	36. 796	0.011
38. 097	38. 083	0.014
38. 451	38. 412	0. 039
38. 796	38.76	0.036
39.619	39. 617	0.002
39.956	39. 926	0.03





EM admittance of PZT bonded to plate (Xu, et al. 2004)





# **Theoretical Analysis**

- Modeling of PZT transducers (2-D generic impedance model)
- Modeling of host structure (Ritz method, polynomial shape functions, Hamilton Principle)





# Assumptions

- Homogeneous and isotropic material
- Thin shell model
- The center plane is stress free
- The in-plane displacements of the center plane are neglected
- The line perpendicular to the center plane remains perpendicular to the center plane after deformation.



# 



#### Stiffness and mass matrices for undamaged shell

$$k_{mn} = \int_{-1-1}^{1-1} \left\{ \frac{Eh^{3}}{12(1-\mu^{2})} \left[ \frac{16}{L^{4}} \frac{\partial^{2} \phi_{m}}{\partial \xi^{2}} \frac{\partial^{2} \phi_{n}}{\partial \xi^{2}} + \frac{\mu}{a^{2}} \frac{16}{L^{2}\pi^{2}} \frac{\partial^{2} \phi_{m}}{\partial \eta^{2}} \frac{\partial^{2} \phi_{n}}{\partial \xi^{2}} \right] + \frac{Eh^{3}}{12(1-\mu^{2})} \left[ \frac{1}{a^{4}} \frac{16}{\pi^{4}} \frac{\partial^{2} \phi_{m}}{\partial \eta^{2}} \frac{\partial^{2} \phi_{n}}{\partial \eta^{2}} + \frac{\mu}{a^{2}} \frac{16}{\pi^{2}L^{2}} \frac{\partial^{2} \phi_{m}}{\partial \xi^{2}} \frac{\partial^{2} \phi_{n}}{\partial \eta^{2}} \right] \right\} a \frac{\pi L}{4} d\xi d\eta \\ + \frac{Eh}{1-\mu^{2}} \frac{1}{a^{2}} \phi_{m} \phi_{n} + \frac{Gh^{3}}{3a^{2}} \frac{16}{\pi^{2}L^{2}} \frac{\partial^{2} \phi_{m}}{\partial \xi \partial \eta} \frac{\partial^{2} \phi_{n}}{\partial \xi \partial \eta} \right\}$$

$$m_{mn} = \int_{-1-1}^{1} \int_{-1-1}^{1} \left[ \frac{\rho h^3}{3L^2} \frac{\partial \phi_m}{\partial \xi} \frac{\partial \phi_n}{\partial \xi} + \frac{\rho h^3}{3a^2 \pi^2} \frac{\partial \phi_m}{\partial \eta} \frac{\partial \phi_n}{\partial \eta} + \rho h \phi_m \phi_n \right] a \frac{\pi L}{4} d\xi d\eta$$



#### Stiffness and mass matrices for damaged shell

$$k_{mn} = \frac{Eh^{3}a\pi L}{3(1-\mu^{2})} \int_{-1}^{1} \int_{-1}^{1} \left[ \frac{1}{L^{4}} \frac{\partial^{2}\phi_{m}}{\partial\xi^{2}} \frac{\partial^{2}\phi_{n}}{\partial\xi^{2}} + \frac{\mu}{a^{2}\pi^{2}L^{2}} \frac{\partial^{2}\phi_{m}}{\partial\xi^{2}} \frac{\partial^{2}\phi_{n}}{\partial\eta^{2}} + \frac{\mu}{a^{2}\pi^{2}L^{2}} \frac{\partial^{2}\phi_{m}}{\partial\eta^{2}} \frac{\partial^{2}\phi_{n}}{\partial\xi^{2}} \right] d\xi d\eta$$
$$- \frac{4(E-E_{d})h^{3}}{3(1-\mu^{2})} \left[ \frac{1}{L^{4}} \frac{\partial^{2}\phi_{m}}{\partial\xi^{2}} \frac{\partial^{2}\phi_{n}}{\partial\xi^{2}} + \frac{\mu}{a^{2}\pi^{2}L^{2}} \frac{\partial^{2}\phi_{m}}{\partial\xi^{2}} \frac{\partial^{2}\phi_{m}}{\partial\eta^{2}} + \frac{\mu}{a^{2}\pi^{2}L^{2}} \frac{\partial^{2}\phi_{m}}{\partial\xi\partial\eta} \frac{\partial^{2}\phi_{n}}{\partial\xi\partial\eta} \frac{\partial^{2}\phi_{n}}{\partial\xi\partial\eta} \right]_{\xi=\xi_{d}} + \frac{\mu}{a^{4}\pi^{4}} \frac{\partial^{2}\phi_{m}}{\partial\eta^{2}} \frac{\partial^{2}\phi_{n}}{\partial\xi^{2}} + \frac{\mu}{a^{2}\pi^{2}L^{2}} \frac{\partial^{2}\phi_{m}}{\partial\eta^{2}} \frac{\partial^{2}\phi_{m}}{\partial\eta^{2}} \frac{\partial^{2}\phi_{n}}{\partial\xi^{2}} + \frac{\mu}{a^{2}\pi^{2}L^{2}} \frac{\partial^{2}\phi_{m}}{\partial\eta^{2}} \frac{\partial^{2}\phi_{m}}{\partial\eta^{2}} \frac{\partial^{2}\phi_{n}}{\partial\xi\partial\eta} \right]_{\xi=\xi_{d}} + \frac{1}{a^{4}\pi^{4}} \frac{\partial^{2}\phi_{m}}{\partial\eta^{2}} \frac{\partial^{2}\phi_{n}}{\partial\eta^{2}} + \frac{3}{4h^{2}a^{2}} \phi_{m}\phi_{n} + \frac{2(1-\mu)}{a^{2}\pi^{2}L^{2}} \frac{\partial^{2}\phi_{m}}{\partial\xi\partial\eta} \frac{\partial^{2}\phi_{n}}{\partial\xi^{2}} + A_{d}$$

$$m_{mn} = \int_{-1-1}^{1} \left[ \frac{\rho h^3}{3L^2} \frac{\partial \phi_m}{\partial \xi} \frac{\partial \phi_n}{\partial \xi} + \frac{\rho h^3}{3a^2 \pi^2} \frac{\partial \phi_m}{\partial \eta} \frac{\partial \phi_n}{\partial \eta} + \rho h \phi_m \phi_n \right] a \frac{\pi L}{4} d\xi d\eta$$



$$G_{k} = \frac{\int_{0}^{L} \int_{0}^{\pi} - \left[\frac{\partial \overline{M}_{1}}{\partial x} + \frac{1}{a} \frac{\partial \overline{M}_{2}}{\partial \theta}\right] \Phi_{k} a d \theta dx}{\mathbf{C}_{k}^{T} \mathbf{M} \mathbf{C}_{k}} = G_{k}^{1} + G_{k}^{2}$$

$$G_{k}^{1} = \frac{\int_{0}^{L} \int_{0}^{\pi} -\left(\frac{\partial \overline{M}_{1}}{\partial x}\right) \Phi_{k} a d\theta dx}{\mathbf{C}_{k}^{T} \mathbf{M} \mathbf{C}_{k}} = \frac{\frac{h + h_{p}}{b_{p}} \frac{a \pi}{L} \sum_{m} \left[ c_{mk} \int_{\eta_{1}}^{\eta_{2}} \left(\frac{\partial \phi_{m}}{\partial \xi}\Big|_{\xi = \xi_{2}} - \frac{\partial \phi_{m}}{\partial \xi}\Big|_{\xi = \xi_{1}}\right) d\eta}{\mathbf{C}_{k}^{T} \mathbf{M} \mathbf{C}_{k}} \cdot \overline{F_{1}} = P_{k} \overline{F_{1}}$$

$$G_{k}^{2} = \frac{\int_{0}^{L} \int_{0}^{\pi} -\left(\frac{1}{a} \frac{\partial \overline{M}_{2}}{\partial \theta}\right) \Phi_{k} a d\theta dx}{\mathbf{C}_{k}^{T} \mathbf{M} \mathbf{C}_{k}} = \frac{\frac{h + h_{p}}{a_{p}} \frac{L}{a \pi} \sum_{m} \left[ c_{mk} \int_{\xi_{1}}^{\xi_{2}} \left(\frac{\partial \phi_{m}}{\partial \eta}\Big|_{\eta = \eta_{2}} - \frac{\partial \phi_{m}}{\partial \eta}\Big|_{\eta = \eta_{1}} \right) d\xi}{\mathbf{C}_{k}^{T} \mathbf{M} \mathbf{C}_{k}} \cdot \overline{F}_{2} = Q_{k} \overline{F}_{2}$$



$$u_{x}(x,\theta,t) = -\frac{h+h_{p}}{2} \frac{\partial \overline{w}(x,\theta,t)}{\partial x}$$
$$v_{\theta}(x,\theta,t) = -\frac{h+h_{p}}{2a} \frac{\partial \overline{w}(x,\theta,t)}{\partial \theta}$$

#### Force-velocity response relation

$$\begin{cases} \dot{u}_{x} \Big|_{x=x_{1}} - \dot{u}_{x} \Big|_{x=x_{2}} \\ a\dot{v}_{\theta} \Big|_{\theta=\theta_{1}} - a\dot{v}_{\theta} \Big|_{\theta=\theta_{2}} \end{cases} = \mathbf{Q} \begin{cases} \overline{F}_{1} \\ \overline{F}_{2} \end{cases} \\ \mathbf{Z}_{str} = \frac{1}{2} \mathbf{Q}^{-1} \end{cases}$$



#### **Admittance of host structure**

$$Q_{11} = j\omega \frac{h+h_p}{L} \sum_{k} \left\{ \frac{P_k}{\omega_k^2 - \omega^2} \sum_{k} \left[ c_{mk} \left( \frac{\partial \phi_m}{\partial \xi} \Big|_{\substack{\xi = \xi_2 \\ \eta = \eta_c}} - \frac{\partial \phi_m}{\partial \xi} \Big|_{\substack{\xi = \xi_1 \\ \eta = \eta_c}} \right) \right] \right\}$$

$$Q_{12} = j\omega \frac{h+h_p}{L} \sum_{k} \left\{ \frac{Q_k}{\omega_k^2 - \omega^2} \sum_{k} \left[ c_{mk} \left( \frac{\partial \phi_m}{\partial \xi} \Big|_{\substack{\xi = \xi_2 \\ \eta = \eta_c}} - \frac{\partial \phi_m}{\partial \xi} \Big|_{\substack{\xi = \xi_1 \\ \eta = \eta_c}} \right) \right] \right\}$$

$$Q_{21} = j\omega \frac{h+h_p}{a\pi} \sum_{k} \left\{ \frac{P_k}{\omega_k^2 - \omega^2} \sum_{k} \left[ c_{mk} \left( \frac{\partial \phi_m}{\partial \eta} \Big|_{\substack{\eta = \eta_2 \\ \xi = \xi_c}} - \frac{\partial \phi_m}{\partial \eta} \Big|_{\substack{\eta = \eta_1 \\ \xi = \xi_c}} \right) \right] \right\}$$

$$Q_{22} = j\omega \frac{h+h_p}{a\pi} \sum_{k} \left\{ \frac{Q_k}{\omega_k^2 - \omega^2} \sum_{k} \left[ c_{mk} \left( \frac{\partial \phi_m}{\partial \eta} \Big|_{\substack{\eta = \eta_2 \\ \xi = \xi_c}} - \frac{\partial \phi_m}{\partial \eta} \Big|_{\substack{\eta = \eta_1 \\ \xi = \xi_c}} \right) \right] \right\}$$